



# UNIVERSITY OF RUHUNA

Faculty of Engineering

End-Semester 3 Examination in Engineering: August 2022

Module Number: ME 3305    Module Name: Modelling and Controlling of Dynamic Systems

[Three Hours]

[Answer all questions, each question carries twelve marks]

Important:

Some necessary equations and a partial table of Laplace transformation pairs have been provided on the question paper. **You may make additional assumptions**, if necessary, by clearly stating them in your answers. Some standard notations may have been used without defining them.

The standard form of a second-order system is  $G(s) = \frac{k\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$ ;

$$T_s = \frac{4}{\zeta\omega_n} \text{ (}\pm 2\% \text{ settling time);}$$

$$\text{Percentage Overshoot} = e^{-\left(\zeta\pi/\sqrt{1-\zeta^2}\right)} \times 100;$$

Q1 a) Figure Q1 shows a mass, spring, and damper system with two masses, two dampers and one spring. The mass  $m_1$  is subjected to an external force  $r(t)$  as shown. The velocity of mass  $m_1$  is  $v_1(t)$  and the velocity of mass  $m_2$  is  $v_2(t)$  as shown.

i. Draw the free body diagram and clearly indicate force vectors acting on each mass. [2 Marks]

ii. Obtain the time domain equations describing **velocity** of mass  $m_1$  and  $m_2$ . [4 Marks]

iii. Obtain the Laplace domain model of the system. [2 Marks]

iv. State any assumptions you made to obtain the above mathematical models. [1 Mark]

b) A system is described by the following transfer function.

$$\frac{Y(s)}{R(s)} = \frac{15(s+1)}{s^2 + 9s + 14}$$

If this system is subjected to a unit step input, obtain the response  $y(t)$  of the system. [3 Marks]

Q2 a) Figure Q2 shows a passive electrical system where  $C = 2\mu F$  and  $R_1 = R_2 = 1M\Omega$ , output of the system is  $v_2(t)$  and input is  $v_1(t)$ .

i. Obtain the transfer function of the system in Laplace domain, in terms of  $R_1$ ,  $R_2$ , and  $C$  [3 Marks]

- ii. State the order of the system. [1 Mark]
- iii. Obtain the steady state gain and time constant of the system. [2 Marks]
- iv. State poles and zeros of the system separately. [1 Mark]
- iv. Obtain the unit step response of the system  $v_2(t)$  in the time domain. [2 Marks]
- b) Transfer function of a system is given by  $G(s) = \frac{a}{s+a}$ . The system was excited by a unit step input at the time  $t = 0$ . Calculate the time  $t$  when the system response is 0.39. [3 Marks]
- Q3 a) The RLC system shown in figure Q3 is subjected to a voltage supply  $\hat{v}(t) = \hat{V} \cos(\omega t)$ . Assume the charge of the capacitor is  $q(t)$ .
- i. State the forcing voltage  $\hat{v}(t)$  in complex exponential form. [1 Mark]
- ii. Obtain the mathematical model of the system in time domain in terms of  $L$ ,  $R$ ,  $C$ ,  $q(t)$  and  $\hat{v}(t)$ . [1 Mark]
- iii. Explain why it is possible to use  $\hat{q}(t) = \hat{Q} e^{j\omega t}$  with same  $\omega$  as  $\hat{v}(t)$  to replace  $\hat{q}(t)$  in (ii) above when the system is in steady state. [1 Mark]
- iv. Obtain an expression to describe complex charge  $\hat{q}(t)$ . [3 Marks]
- v. The impedance is defined as  $\hat{Z} = \hat{v} / \hat{i}$  where  $\hat{v}$  is voltage and  $\hat{i}$  is current. Obtain the reactance of the system. [3 Marks]
- b) Harmonic oscillation of a system is described as superposition of the following two models.
- $$\hat{x}_1 = A_1 e^{j(\omega t + \varphi_1)} \quad , \quad \hat{x}_2 = A_2 e^{j(\omega t + \varphi_2)}$$
- i. Draw the phasor diagram to show superposition of two harmonics. [1 Mark]
- ii. Obtain the amplitude  $A$  and the phase  $\varphi$  of the vibration of entire system. [2 Marks]
- Q4 The rotational mass, damper, and spring system shown in figure Q4 is subjected to  $\tau(t)$  torque. The angle of rotation of the mass is  $\theta(t)$ .

Notes:

- Inverse of Matrix  $A$  is  $A^{-1} = \frac{\text{Adjoint}(A)}{\text{Determinant}(A)}$
- Transpose of cofactor matrix is defined as adjoint matrix.
- $C_{ij}$  th element of cofactor matrix is defined as  $C_{ij} = (-1)^{(i+j)} \times M_{ij}$  where,  $M_{ij}$  is the respective Minor).

4. Minor  $M_{ij}$  of  $2 \times 2$  matrix can be obtained by removing  $i^{\text{th}}$  row and  $j^{\text{th}}$  column from  $2 \times 2$  matrix.

- a. Draw the free body diagram, indicating all the forces/torques acting on the mass. [1 Mark]
  - b. Obtain the state space model of the system taking  $\theta$  and  $\dot{\theta}$  as states. [2 Marks]
  - c. Taking  $J = 2 \text{ kgm}^2$ ,  $B = 8 \text{ Ns/m}$ ,  $K = 6 \text{ N/m}$ ,  $\tau(t) = u = 0$  obtain the characteristic equation using the state space model obtained above. [2 Marks]
  - d. Obtain the natural frequency and damping ratio of the system. [2 Marks]
  - e. Obtain the state transition matrixes  $\Phi(s)$  and  $\Phi(t)$ . [2 Marks]
  - f. Based on the results of (e), state whether this system is stable or not and give reasons for your answer. [4 Marks]
- [1 Mark]

Q5 a) A nonlinear dynamics of a system is given as  $\dot{x} = F(x) = \sin(x)$ .

- i. Determine the stability of the system around operating point  $x_1 = 0 \text{ rad}$ . [2 Marks]
- ii. Determine the stability of the system around operating point  $x_2 = \pi \text{ rad}$ . [2 Marks]
- iii. Draw the graph showing  $x$  vs  $F(x)$  and use the results above to indicate the direction of motion of the system when  $x$  is between  $-2\pi \text{ rad}$  to  $+2\pi \text{ rad}$  using arrow heads on the  $x$  axis. [2 Marks]

b) A dynamic system is described by,

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} -2 & 0 \\ 3 & -5 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 1 \\ 0 \end{bmatrix} [u]$$

$$y = [1 \quad -1] \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$$

- i. Determine whether this system is controllable. [2 Marks]
  - ii. Determine whether this system is observable. [2 Marks]
- c) State advantages of state space system analysis over transfer function model analysis for dynamic systems. [2 Marks]

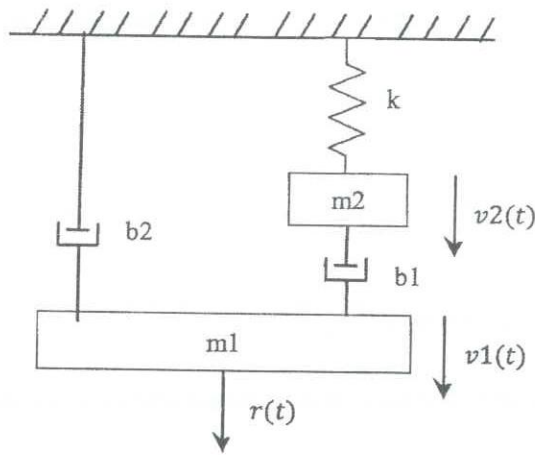


Figure Q1

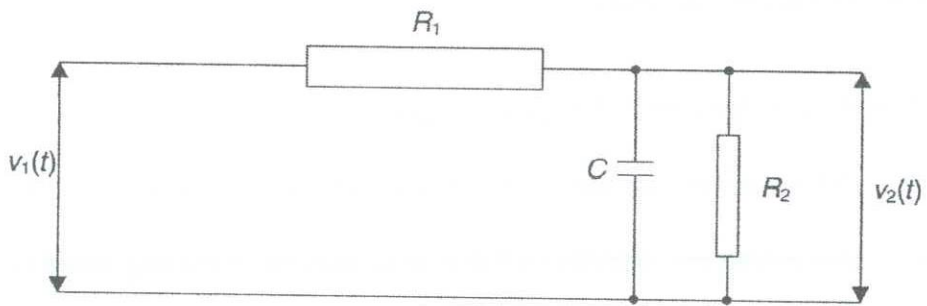


Figure Q2

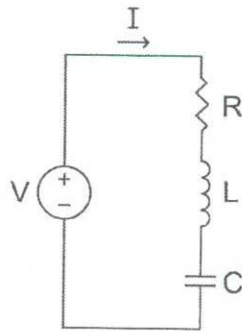


Figure Q3

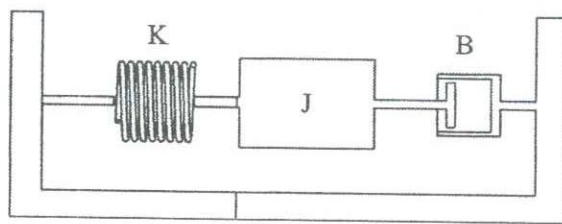


Figure Q4

Laplace Transforms Table

$f(t) = L^{-1}\{F(s)\}$	$F(s)$	$f(t) = L^{-1}\{F(s)\}$	$F(s)$
$a \quad t \geq 0$	$\frac{a}{s} \quad s > 0$	$\sin \omega t$	$\frac{\omega}{s^2 + \omega^2}$
$at \quad t \geq 0$	$\frac{a}{s^2}$	$\cos \omega t$	$\frac{s}{s^2 + \omega^2}$
$e^{-at}$	$\frac{1}{s+a}$	$\sin(\omega t + \theta)$	$\frac{s \sin \theta + \omega \cos \theta}{s^2 + \omega^2}$
$te^{-at}$	$\frac{1}{(s+a)^2}$	$\cos(\omega t + \theta)$	$\frac{s \cos \theta - \omega \sin \theta}{s^2 + \omega^2}$
$\frac{1}{2}t^2e^{-at}$	$\frac{1}{(s+a)^3}$	$t \sin \omega t$	$\frac{2\omega s}{(s^2 + \omega^2)^2}$
$\frac{1}{(n-1)!}t^{n-1}e^{-at}$	$\frac{1}{(s+a)^n}$	$t \cos \omega t$	$\frac{s^2 - \omega^2}{(s^2 + \omega^2)^2}$
$e^{at}$	$\frac{1}{s-a} \quad s > a$	$\sinh \omega t$	$\frac{\omega}{s^2 + \omega^2} \quad s >  \omega $
$te^{at}$	$\frac{1}{(s-a)^2}$	$\cosh \omega t$	$\frac{s}{s^2 + \omega^2} \quad s >  \omega $
$\frac{1}{b-a}(e^{-at} - e^{-bt})$	$\frac{1}{(s+a)(s+b)}$	$e^{-at} \sin \omega t$	$\frac{\omega}{(s+a)^2 + \omega^2}$
$\frac{1}{a^2}[1 - e^{-at}(1 + at)]$	$\frac{1}{s(s+a)^2}$	$e^{-at} \cos \omega t$	$\frac{s+a}{(s+a)^2 + \omega^2}$
$t^n$	$\frac{n!}{s^{n+1}} \quad n = 1, 2, 3$	$e^{at} \sin \omega t$	$\frac{\omega}{(s-a)^2 + \omega^2}$
$t^n e^{at}$	$\frac{n!}{(s-a)^{n+1}} \quad s > a$	$e^{at} \cos \omega t$	$\frac{s-a}{(s-a)^2 + \omega^2}$
$t^n e^{-at}$	$\frac{n!}{(s-a)^{n+1}} \quad s > a$	$1 - e^{-at}$	$\frac{a}{s(s+a)}$
$\sqrt{t}$	$\frac{\sqrt{\pi}}{2s^{3/2}}$	$\frac{1}{a^2}(at - 1 + e^{-at})$	$\frac{1}{s^2(s+a)}$
$\frac{1}{\sqrt{t}}$	$\sqrt{\frac{\pi}{s}} \quad s > 0$	$f(t - t_1)$	$e^{-t_1 s} F(s)$
$g(t) \cdot p(t)$	$G(s) \cdot P(s)$	$f_1(t) \pm f_2(t)$	$F_1(s) \pm F_2(s)$
$\int f(t) dt$	$\frac{F(s)}{s} + \frac{f^{-1}(0)}{s}$	$\delta(t)$ Unit impulse	1 all $s$
$\frac{df}{dt}$	$sF(s) - f(0)$	$\frac{d^2 f}{df^2}$	$s^2 F(s) - sf(0) - f'(0)$
$\frac{d^n f}{dt^n}$	$s^n F(s) - s^{n-1} f(0) - s^{n-2} f'(0) - s^{n-3} f''(0) - \dots - f^{(n-1)}(0)$		