



UNIVERSITY OF RUHUNA

Faculty of Engineering

End-Semester 6 Examination in Engineering: December 2018

Module Number: EE6302

Module Name: Control system Design

[Three Hours]

[Answer all questions, each question carries 15 marks]

All notations have their usual meanings. Formulas you may require is given in page 4 and Laplace transform table is given in Table No 1, page 7.

- Q1. a) i) Sketch a suitable time response to illustrate the *rise time*, *overshoot* and *settling time*.
- ii) Give expressions for the time domain specifications given in part a).i) for the system given in Figure Q1(a).
- iii) Sketch the allowable region in the S-Plane for the locations of poles of a second order unit step response whose time domain specifications are to be kept as; *rise time*  $\leq a$  s, *overshoot* %  $\leq b$ % and *settling time* (1% criteria)  $\leq c$  s, where *a*, *b* and *c* are real numbers.

[5.5 Marks]

- b) You are asked to design a simple speed control system of a DC motor as shown in Figure Q1 (b). The motor is running at no-load with negligible load torque. For a unit impulse armature voltage of the DC motor  $v_a(t)$ , the angular speed  $\omega_m(t)$  is expressed as

$$\omega_m(t) = 10[e^{-2t} - e^{-10t}]$$

- i) Obtain the transfer function of the DC motor and hence, find the equivalent circuit parameters of the DC motor  $R_a$ ,  $L_a$ ,  $J$ , and  $b$  shown in Figure Q1 (b).
- ii) Calculate the value of the PE actuator constant  $A$  at which the DC motor speed control system undergoes critically damped stage.
- iii) Determine the factor by which  $A$  should be multiplied to reduce the damping ratio( $\xi$ ) from critical stage to 0.6.
- iv) With this motor speed control system, it is required to track the speed reference of  $200 \text{ rads}^{-1}$ . Find the rated speed of the DC motor, in order that the motor speed control system be a stable system with  $\xi = 0.6$ .
- v) Hence, discuss the reference tracking capability of this DC motor speed control system.

[9.5 Marks]

Directions : Those who have not answered part b)i), can find the answers for the rest of parts in terms of  $R_a$ ,  $L_a$ ,  $J$ , and  $b$ .

- Q2. a) i) Write the general form of the matrix equations so that a system is represented in a state-space model. Name the matrices in your matrix equations.
- ii) Derive the transfer function of the system using the matrix equations written in part a) i).
- iii) State the Routh's necessary and sufficient conditions to have a stable system.

[4.0 Marks]

- b) It is required to evaluate the stability of a plant  $G(s)$ , which consists of two subsystems  $G_1$  and  $G_2$ , connected in series. The block diagram of  $G_1(s)$  is shown in Figure Q2 (b1) and Figure Q2 (b2) illustrates the locations of the poles and zeroes of  $G_2(s)$  in the s-plane.

- i) If the state vector of the plant can be denoted as  $[x_1 \ x_2 \ \dots \ x_n]$  and the block diagram has 6, 5, -4 and  $K$  for  $k_1, k_2, k_3$  and  $k_4$ , respectively, formulate the state-space model of the subsystem  $G_1(s)$  shown in Figure Q2 (b1).
- ii) Hence, obtain the transfer function of  $G_1(s)$ .
- iii) Find the range of  $K$  for which the plant  $G(s)$  is stable.

[6.0 Marks]

- c) i) What are the advantages of feedback control when compared with open-loop control?
- ii) Figure Q2 (c1) and Q2 (c2) illustrate the block diagrams of an open-loop and a closed loop control system of a plant. The transfer function of the plant is

$$G(s) = \frac{A}{Ts + 1}$$

The controller is a proportional controller with gain  $K$ .

Derive expressions for steady state values of the plant output for the open loop control system and the closed loop control system when  $W(s) = \frac{w}{s}$  and  $R(s) = \frac{r}{s}$ . Here,  $w$  and  $r$  are constants.

Using the expressions you have derived, comment on the impact of disturbance on the response of the open and closed loop control systems.

[5.0 marks]

- Q3. a) i) State the definition of the root locus.
- ii) Explain how you find the system gain to operate the closed loop transient response at a given percent overshoot using root locus.

[4.0 marks]

- b) Consider the unity feedback system shown in Figure Q3 where

$$G(s) = \frac{K}{(s + 10)(s^2 + 4s + 5)}$$

- i) Plot the root locus for this system.
- ii) Find the range of gain  $K$  where the closed loop system is stable.
- iii) Find the value of gain  $K$  that yields 15% overshoot in the closed loop step response.
- iv) Find the closed loop poles of the system for the gain you have found in part b)iii).
- v) Evaluate the accuracy of the second order approximation of the closed loop transfer function for the gain you have found in part b)iii).
- vi) Estimate the settling time and the peak time of the closed loop system operating at 15% overshoot.
- vii) Calculate the steady state error of the gain adjusted system for a unit step input.
- viii) Design a suitable compensator for the gain adjusted system with  $K = 56.89$  to improve the steady state error by a factor of 10.

[11.0 marks]

- Q4. a)
  - i) Define the terms phase margin and gain margin associated with Bode plots.
  - ii) Explain how you evaluate the stability of a closed-loop system using the frequency response of an open-loop system.

[4.0 marks]

- b) Consider the unity feedback system shown in Figure Q4 where

$$G(s) = \frac{K}{(s + 2)(s + 3)(s + 12)}$$

- i) Find the steady state error of the closed-loop system for a unit step input when  $K = 1$  and  $G_c(s) = 1$ .
- ii) Find the required gain to improve the steady state error of the closed-loop system by a factor of 20.
- iii) Sketch the bode magnitude and bode phase responses of  $G(s)$  for the gain  $K$  you have found in part b).ii).
- iv) Evaluate the stability of the closed loop system for the gain you have found in part b).ii) using the frequency response of the open-loop system.
- v) Using the frequency response, find the range of gain  $K$  where the closed-loop system is stable.

- vi) Design a lag compensator for the gain adjusted system with  $K= 658.2$  to improve the percent overshoot to 15%. Use the frequency response design technique.

[11.0 marks]

Formulas you may require:

$$\phi_M = \tan^{-1} \left( \frac{2\xi}{\sqrt{-2\xi^2 + \sqrt{1 + 4\xi^4}}} \right)$$

$$\omega_{BW} = \frac{4}{T_s \xi} \sqrt{(1 - 2\xi^2) + \sqrt{4\xi^4 - 4\xi^2 + 2}}$$

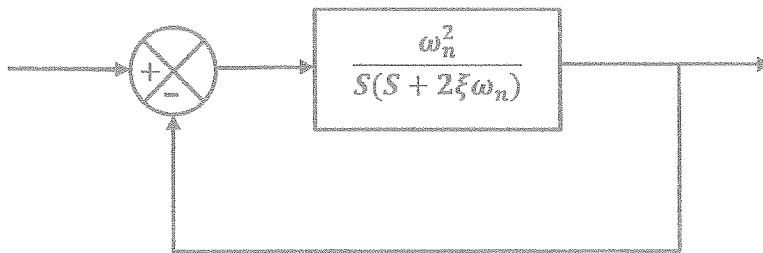


Figure Q1 (a).

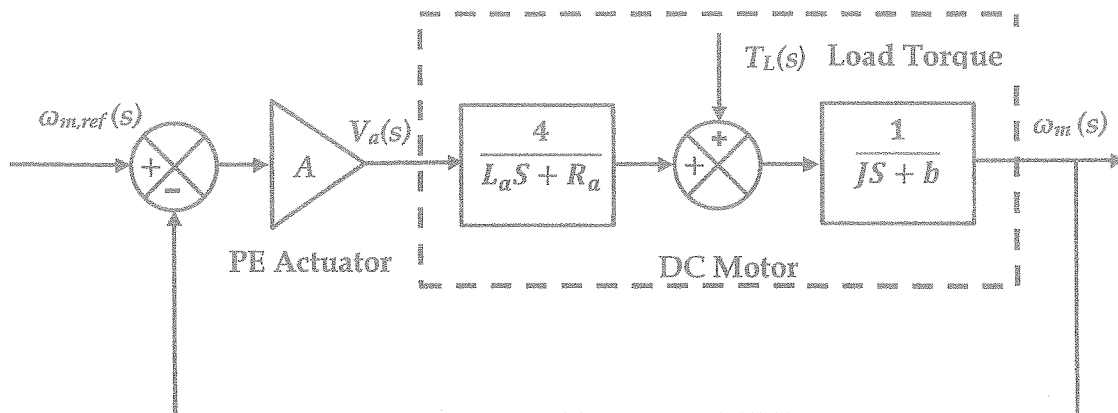


Figure Q1 (b).

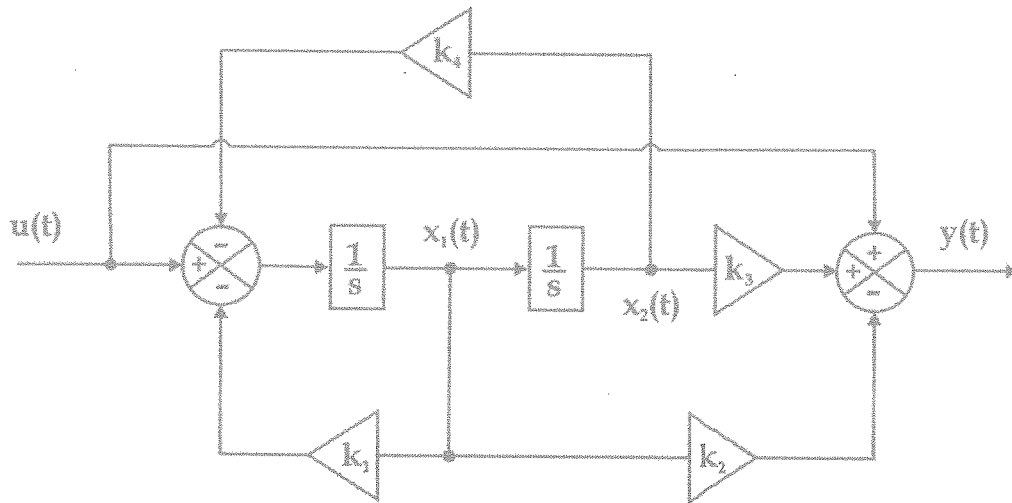


Figure Q2 (b1).

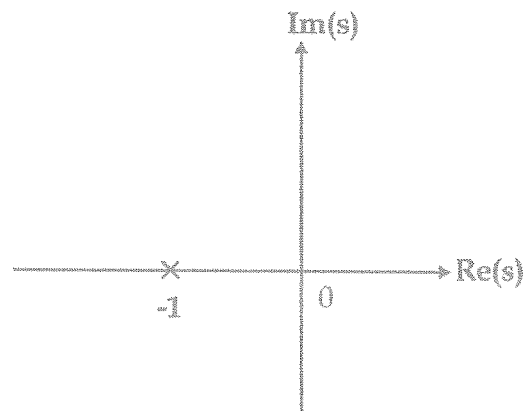


Figure Q2 (b2).

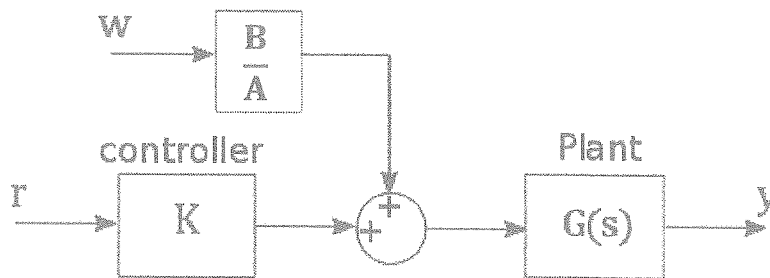


Figure Q2 (c1) Open-loop control system.

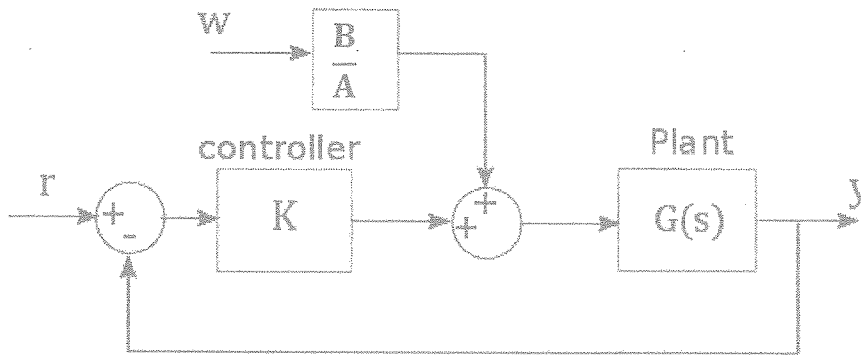


Figure Q2 (c2) Closed-loop control system.

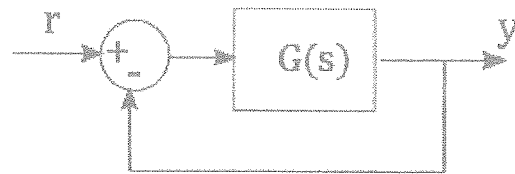


Figure Q3.

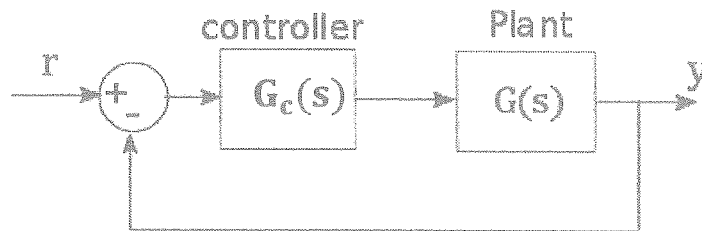


Figure Q4.

Table No 1: Laplace Transform Table

$f(t)$	$F(s)$
$\delta(t)$	1
$u(t)$	$\frac{1}{s}$
$e^{-at}$	$\frac{1}{s+a}$
$\sin \omega t$	$\frac{\omega}{s^2 + \omega^2}$
$\cos \omega t$	$\frac{s}{s^2 + \omega^2}$
$e^{-at} \sin \omega t$	$\frac{\omega}{(s+a)^2 + \omega^2}$
$e^{-at} \cos \omega t$	$\frac{s+a}{(s+a)^2 + \omega^2}$
$\frac{df(t)}{dt}$	$sF(s) - f(0)$
$\int f(t)dt$	$\frac{F(s)}{s} + \frac{1}{s} \int (f(t)dt)  _{t=0}$