



UNIVERSITY OF RUHUNA

Faculty of Engineering

End-Semester 5 Examination in Engineering: August 2014

Module Number: CE5217

Module Name: Structural Analysis III

[Three Hours]

[Answer all questions. Each question carries 12 marks]

All Standard Notations denote their regular meanings

Q1. a) What is the virtual work equation for yield line theory? Clearly define the parameters in the equation. [2 marks]

b) Figure Q1(a) shows the isotropic slab is subjected to a vertical downward load of intensity ' p ' per unit area. Find required moment capacity of the slab. [3 marks]

b) An isotropically reinforced rectangular slab shown in Figure Q1(b) is simply supported on all four edges. The slab carries a uniformly distributed load of 10 kN/m².

1. Draw possible yield line pattern at collapse.
2. Determine the collapse load corresponding to above yield line pattern.
3. Find the collapse load factor.

[7 marks]

Q2. A thin rectangular plate of x and y side dimension ' a ', ' b ' and thickness ' t ' is shown in Figure Q2. The plate is simply supported along all four edges. It is subjected to a vertical downward load of intensity, $p(x, y) = p_0 \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b}$. The p_0 is a constant and the m, n are positive integers.

a) Show that the trial solution for displacement $w = w_0 \sin \frac{m\pi x}{a} \sin \frac{n\pi y}{b}$ satisfies the governing differential equation for thin plate when $w_0 = \frac{p_0}{\pi^4 D \left(\frac{m^2}{a^2} + \frac{n^2}{b^2} \right)^2}$, here

$$D = \frac{Et^3}{12(1-\nu^2)}$$

is flexural rigidity of the plate.

[5 marks]

b) Show that the above solution satisfies the relevant displacement and stress boundary conditions.

[2 marks]

- c) Determine the maximum deflection and its position, when $m=2$, $n=2$, $a=b=1$ m, $t = 5$ mm and $p_0 = 300$ kPa. The Young's modulus (E) and the poisson's ratio (ν) of the plate material are 210 GPa and 0.3, respectively.

[5 marks]

Use the Governing differential equation and moment equation (with usual notation and sign convention) as shown below

$$\frac{\partial^4 w}{\partial x^4} + 2 \frac{\partial^2 w}{\partial x^2 \partial y^2} + \frac{\partial^4 w}{\partial y^4} = \frac{q}{D}$$

$$M_x = -D \left(\frac{\partial^2 w}{\partial x^2} + \nu \frac{\partial^2 w}{\partial y^2} \right) \quad \text{and} \quad M_y = -D \left(\frac{\partial^2 w}{\partial y^2} + \nu \frac{\partial^2 w}{\partial x^2} \right)$$

- Q3. A flat circular plate of radius b , is simply-supported concentrically by a tube of radius a , as shown in Figure Q3. If the internal portion of the plate is subjected to a uniform pressure p per unit area. Show that the central deflection δ of the plate is given by

$$\delta = \frac{pa^4}{64D} \left\{ 3 + 2 \left(\frac{a}{b} \right)^2 \left(\frac{1-\nu}{1+\nu} \right) \right\}$$

Use the circular plate differential equation and moment equation (with usual notation and sign convention) as shown below.

$$\frac{d}{dr} \frac{1}{r} \frac{d}{dr} \left(r \frac{dw}{dr} \right) = \frac{F}{D} \quad \text{and} \quad M_r = D \left(\frac{d^2 w}{dr^2} + \nu \frac{dw}{r dr} \right)$$

where $D = \frac{Et^3}{12(1-\nu^2)}$ and F is the shearing force / unit circumferential length.

[12 marks]

- Q4. A spherical fuel oil tank of radius R sits on an annular ring support, as shown in Figure Q4. The oil density is ρ and the oil level is a distance bR from the crown of the tank. The self-weight of the tank can be neglected.

- a) Drive the expressions for the meridional stress in the sphere.

[10 marks]

- b) Determine the meridional stress just above the support ring and hence find the value of b for which this stress is numerically a maximum.

[2 marks]

Use the Membrane force equations for shell (with usual notation and sign convention) as shown below.

$$P_\varphi r_1 r - N_\theta r_1 \cos \varphi + \frac{\partial(rN_\varphi)}{\partial \varphi} = 0$$

$$P_r = \frac{N_\varphi}{r_1} + \frac{N_\theta}{r_2}$$

- Q5. a) From the membrane force equations show that the membrane stresses in a conical shell (with usual notation and sign convention) are given by

$$N_{\theta} = P_r S \tan \alpha$$
$$N_s = \frac{1}{S} \int (P_r S \tan \alpha - P_s S) ds$$

[3 marks]

- b) A conical shape shell is partially submerged in water of density ρ , as shown in Figure Q5. You can assume that the stresses due to self-weight of the shell are negligible. Determine the expressions for the membrane forces in conical part above and below the water level.

[9 marks]

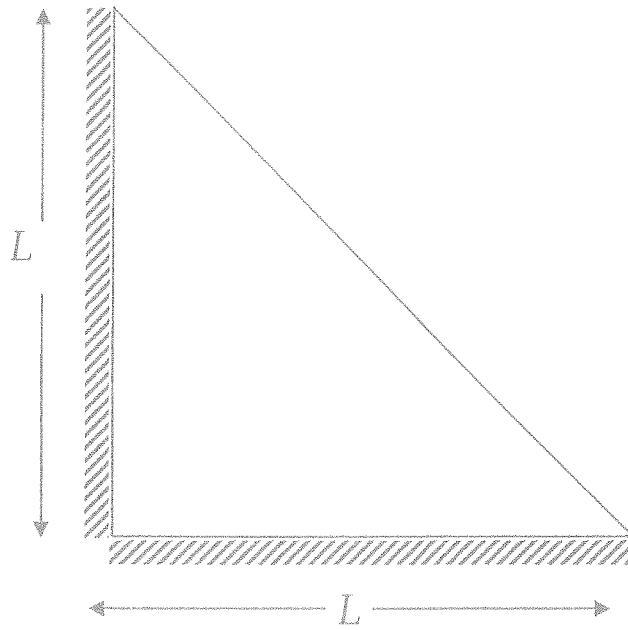


Figure Q1(a)

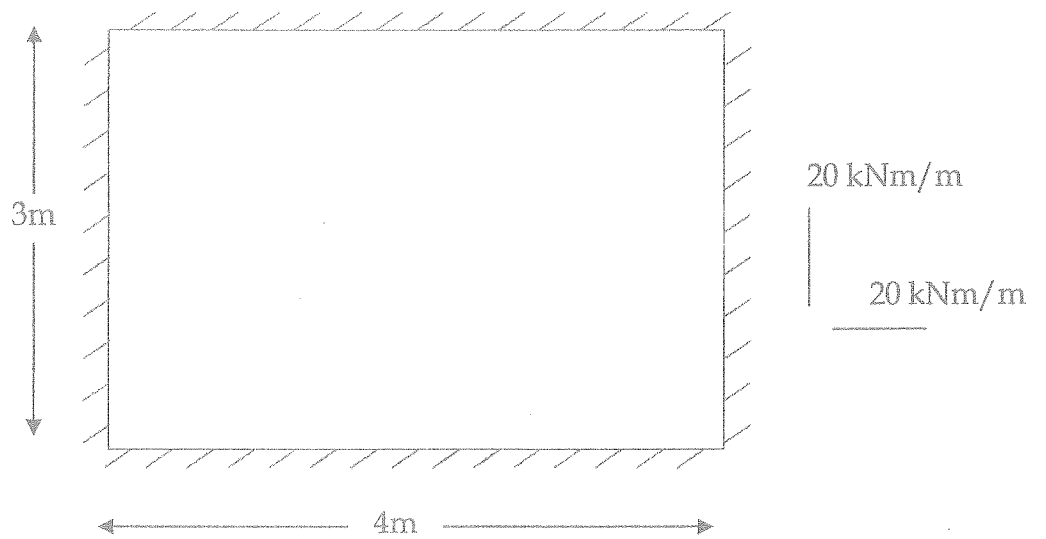


Figure Q1(b)

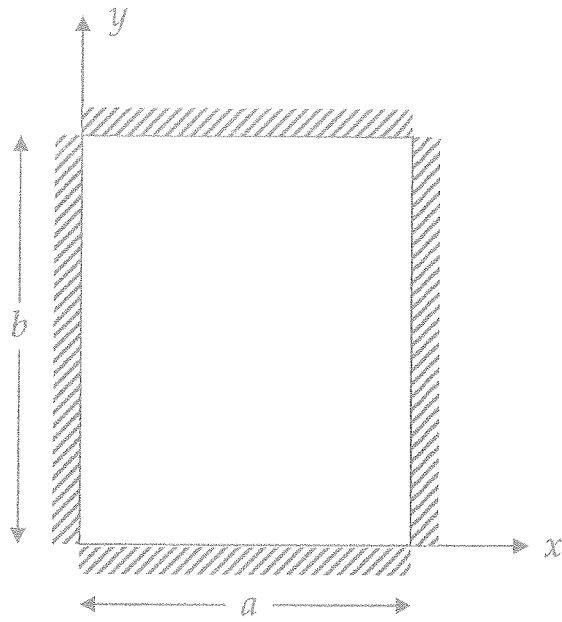


Figure Q2

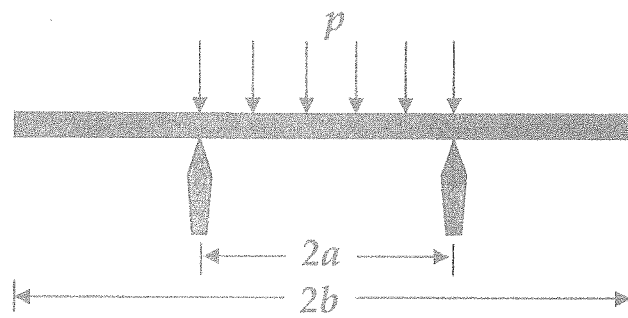


Figure Q3

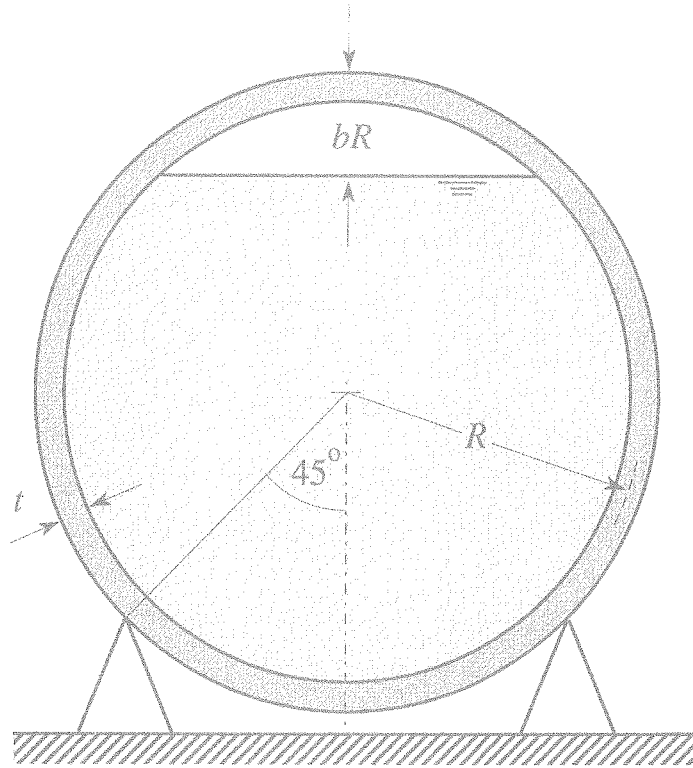


Figure Q4

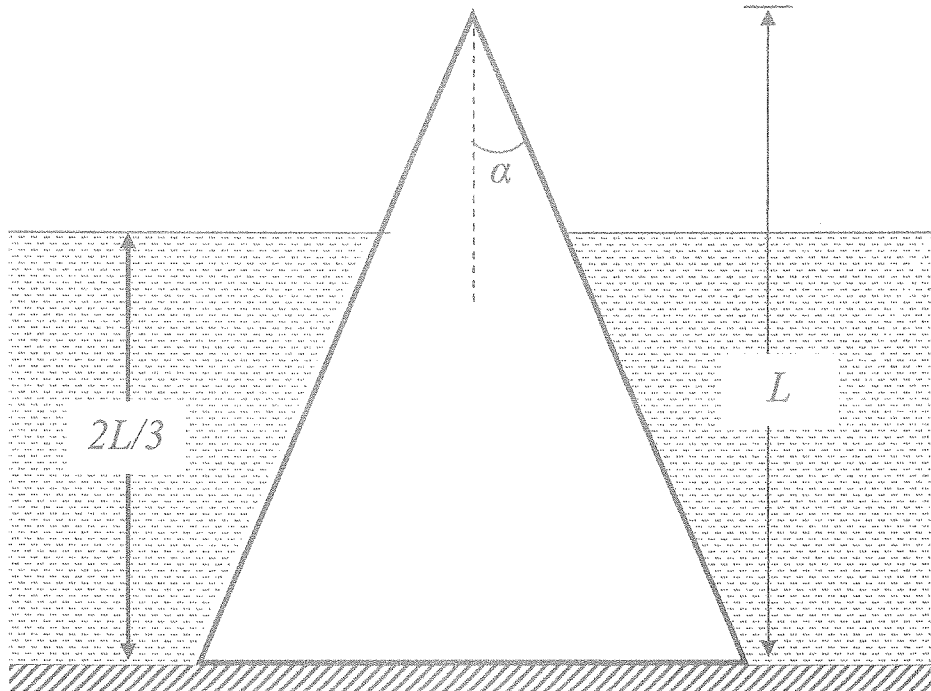


Figure Q5