



FACULTY



UNIVERSITY OF RUHUNA

Faculty of Engineering

End-Semester 3 Examination in Engineering: March 2021

Module Number: ME 3305 Module Name: Modelling and Controlling of Dynamic Systems

[Three Hours]

[Answer all questions, each question carries twelve marks]

Important:

Some necessary equations and a partial table of Laplace transformation pairs have been provided on the question paper. You may make additional assumptions, if necessary, by clearly stating them in your answers. Some standard notations may have been used without defining them.

The standard form of a second-order system is $G(s) = \frac{k\omega_n^2}{S^2 + 2\zeta\omega_n S + \omega_n^2}$;

 $T_s = \frac{4}{\zeta \omega_n}$ (±2% settling time);

Percentage Overshoot = $e^{-(\zeta \pi / \sqrt{1 - \zeta^2})} \times 100$;

- Q1. a) Figure Q1 shows a circuit with a resistor, a capacitor and an inductor. The voltage across a capacitor is described by $v_c(t)$ and the input voltage from the battery is v(t). Current in network is i(t).
 - i. Describe the $v_c(t)$ in terms of charge q(t) of the capacitor and other suitable parameters.

[1 Mark]

ii. Describe the $v_c(t)$ in terms of current through the capacitor and other suitable parameters.

[1 Mark]

iii. Develop a mathematical model describing behavior of the circuit relating input voltage from the battery and output voltage across the capacitor in time domain.

[2 Marks]

iv. Obtain the transfer function of the system in Laplace domain

[2 Marks]

v. State any assumptions you made to obtain answer for (iv) above.

[1 Mark]

- b) A system is described by $\frac{dc(t)}{dt} + 2c(t) = r(t)$ where c(t) is output and r(t) is input.
 - i. Obtain the transfer function of the system in Laplace domain and state any assumptions.

[1 Mark]

ii. If this system was excited by a unit step, obtain the output of the system in Laplace domain.

[1 Mark]

iii. Obtain the time domain response of the system using (ii) above.

[2 Marks]

iv. Plot the time domain response indicating important points. (Graph sheets are not necessary. Use free hand sketch.)

[1 Mark]

- The rotational mass, damper and spring system is shown in Figure Q2. The mass Q2. a) of the system with the polar moment of inertia J was excited by an external torque of T(t) Nm. The rotational spring constant and the damping constant are given by K and B respectively.
 - i. Obtain the transfer function of the system and transform it into Laplace domain assuming output of the system is angular position of the mass heta(t)and input is the torque T(t).

[2 Marks]

ii. Obtain the natural frequency and damping ratio of the system.

[2 Marks]

iii. If this system must have settling time of 3 seconds and maximum of 30% overshoot, under a unit step torque input, find the values of polar moment of inertia and damping constant of the system when the rotational spring constant is 3 Nm/rad.

[3 Marks]

iv. Obtain the poles and zeros of the system and state them separately.

[2 Marks]

Transfer function of a system is given by $G(s) = \frac{a}{s+a}$. The system was excited by a unit step input at time t = 0. Obtain the response of the system at the time $t = \frac{1}{a}$.

[3 Marks]

- The mass-damper system shown in the figure Q3-Q4 is subjected to the forcing Q3. a) function $\hat{f}(t) = \hat{F}.\cos(\omega t)$. The displacement of mass is $\hat{x}(t)$.
 - i. State the forcing function in complex exponential form.

[1 Mark]

ii. Obtain the mathematical model describing motion of the mass in time domain.

[1 Mark]

iii. Explain why it is possible to use $\widehat{x}(t) = Ae^{j\omega t}$ with same ω as $\widehat{f}(t)$ to replace $\hat{x}(t)$ in (ii) above when the system is in steady state.

[1 Mark]

iv. Obtain an expression to describe complex displacement $\hat{x}(t)$.

[3 Marks]

The mechanical **impedance** is defined as $\hat{z} = f/_{\widehat{\mathcal{V}}}$ where $\hat{\mathcal{V}}$ is complex velocity. Obtain the mechanical reactance.

Harmonic oscillation of a system is described as superposition of following two

$$\hat{x}_1 = A_1 e^{j(\omega t + \varphi_1)}$$
 $\hat{x}_2 = A_2 e^{j(\omega t + \varphi_2)}$

i. Draw the phasor diagram to show superposition of two harmonics.

[1 Mark]

ii. Obtain the amplitude A and phase φ of the vibration of entire system.

[2 Marks]

Q4. The mass-damper system shown in figure Q3-Q4 is subjected to $\hat{f}(t) = u$

Notes:

- Inverse of Matrix A is A⁻¹ = Adjoint (A)
 Determinant(A)

 Transpose of cofactor matrix is defined as adjoint matrix.
- 3. C_{ij} th element of cofactor matrix is defined as $C_{ij} = (-1)^{(i+j)} \times M_{ij}$ where, M_{ij} is the respective Minor).
- 4. Minor M_{ij} of 2 × 2 matrix can be obtained by removing i^{th} row and j^{th} column from 2×2 matrix.
- Draw the free body diagram, indicating forces acting on the mass. a.

[1 Mark]

b. Obtain the state space model of the system taking x and \dot{x} as states.

[2 Marks]

Taking m = 2 kg, B = 8 Ns/m, K = 6 N/m, $\hat{f}(t) = u = 0$ obtain the characteristic C. equation using the state space model obtained above.

[2 Marks]

Obtain the natural frequency and damping ratio of the system comparing your answer to (c) with standard second order form.

[2 Marks]

e. Obtain the state transition matrixes $\emptyset(s)$ and $\emptyset(t)$.

[4 Marks]

f. Based on the results of (e), state whether this system is stable or not and give reasons for your answer.

[1 Mark]

- Q5. a) A nonlinear system dynamics is given as $\dot{x} = F(x) = \cos(x)$.
 - Determine the stability of the system around operating point $x_1 = \frac{\pi}{2}$ rad.

ii. Determine the stability of the system around operating point $x_2 = \frac{3\pi}{2}$ rad.

Draw the graph showing x vs F(x) and use the results above to indicate the direction of motion of system when x is between -2π rad to $+2\pi$ rad using arrow heads on the x axis.

[2 Marks]

b) A dynamic system is given by $\dot{x} = F(x) = rx(1-x)$ where r is a positive real valued constant.

i. Find the fixed points (points where velocity is zero) of the system.

ii. Find the Jacobian function of the system.

[1 Mark]

iii. Find the eigenvalues of the Jacobian function.

[1 Mark]

iv. State the stability of each fixed point with reasons.

[2 marks]

[2 Mark]

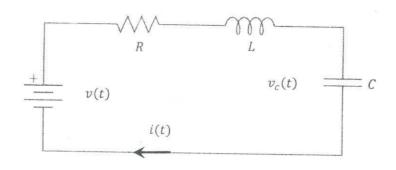


Figure Q1

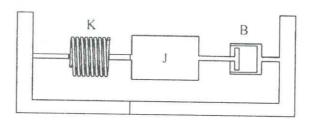


Figure Q2

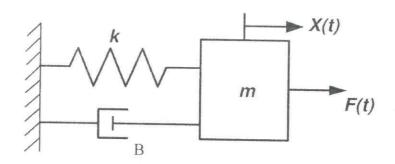


Figure Q3 - Q4

	Laplacetrans	forms - Table	
$f(t) = L^{-1}{F(s)}$	F(s)	$f(t) = L^{-1}\{F(s)\}$	F(s)
$a t \ge 0$	$\frac{a}{s}$ $s > 0$	sin ω t	$\frac{\omega}{s^2 - \omega^2}$
at $t \ge 0$	a s ²	coswt	$\frac{s}{s^2 - \omega^2}$
e-at	$\frac{1}{s-a}$	$\sin(\omega t - \theta)$	$\frac{s\sin\theta + \omega\cos\theta}{s^2 - \omega^2}$
te−a:	$\frac{1}{(s-a)^2}$	$\cos(\omega t - \theta)$	$\frac{s\cos\theta - \omega\sin\theta}{s^2 - \omega^2}$
$\frac{1}{2}t^2e^{-\alpha t}$	$\frac{1}{(s-a)^3}$	t sin wt	$\frac{2\omega s}{(s^2 + \omega^2)^2}$
$\frac{1}{(n-1)!}t^{n-1}e^{-\alpha t}$	$\frac{1}{(s-a)^n}$	tcoswt	$\frac{s^2 - \omega^2}{(s^2 - \omega^2)^2}$
e ^{sst}	$\frac{1}{s-c} \qquad s > \alpha .$	sinh wt	$\frac{\omega}{s^2 - \omega^2} \qquad s > \omega $
te ^{ar}	$\frac{1}{(s-a)^2}$	cosh ωt	$\frac{s}{s^2 - \omega^2} \qquad s > \omega $
$\frac{1}{b-a}\left(e^{-at}-e^{-bt}\right)$	$\frac{1}{(s-a)(s-b)}$	e ^{−at} sin ωt	$\frac{\omega}{(s+\alpha)^2+\omega^2}$
$\frac{1}{\alpha^2}[1-e^{-\alpha t}(1-\alpha t)]$	$\frac{1}{s(s-a)^2}$	e ^{−at} cosωt	$\frac{s-a}{(s-a)^2-\omega^2}$
Fr	$\frac{n!}{s^{n-1}} \qquad n = 1, 2, 3$	e ^{at} sin wt	$\frac{\omega}{(s-a)^2+\omega^2}$
tu ⁶ at	$\frac{n!}{(s-a)^{n-1}} s > a$	e ^{ac} cos.wt	$\frac{s-\alpha}{(s-\alpha)^2-\omega^2}$
t ⁿ e ^{-at}	$\frac{n!}{(s-a)^{n-1}} s > a$	$1 - e^{-\alpha t}$	$\frac{a}{s(s-a)}$
\sqrt{t}	$\frac{\sqrt{\pi}}{2s^{3/2}}$	$\frac{1}{a^2}(at-1-e^{-at})$	$\frac{1}{s^2(s+a)}$
$\frac{1}{\sqrt{\dot{t}}}$	$\sqrt{\frac{\pi}{s}}$ $s > 0$	$f(t-t_1)$	$e^{-t_{\mathcal{L}}s}F(s)$
$g(t) \cdot p(t)$	$G(s) \cdot P(s)$	$f_1(t) \equiv f_2(t)$	$F_1(s) \pm F_2(s)$
$\int f(t)dt$	$\frac{F(s)}{s} + \frac{f^{-1}(0)}{s}$	δ(t) unit impulse	1 alls
a'f dt	sF(s) - f(0)	$\frac{d^2f}{df^2}$	$s^2F(s) - sf(0) - f'(0)$
$\frac{d^nf}{dt^n}$	$s^n F(s) - s^{n-1} f(0) - s^{n-1}$		$\cdots - f^{n-1}(0)$